

## SDNL111 - Impact of two beams

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### Summary:

This problem is a problem of impact of two beams in traction and compression. A first free beam is animated an initial speed parallel with the axis of the two beams and comes to run up against one second embedded against its base. Non-linearity comes from the conditions of contact between the two structures. This test comprises an analytical solution of reference.

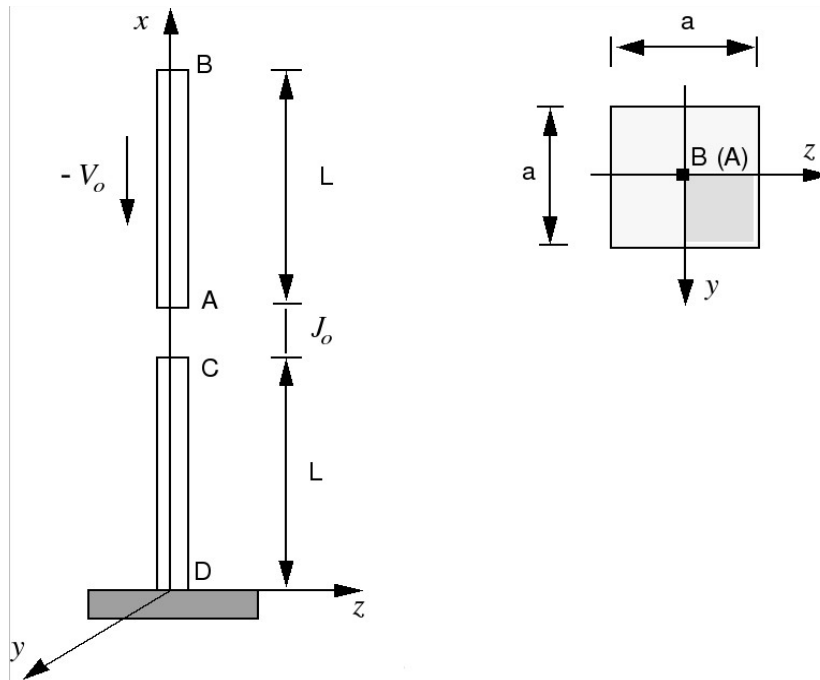
Initially, one uses a transitory analysis by modal recombination of a non-linear system made up by structures of beams (modelings A and B).

The beams are discretized by finite elements of type `POU_D_T`. Operators `DEFI_OBSTACLE` [U4.44.21] and `DYNA_TRAN_MODAL` [U4.53.21] are tested. The variations with the values of reference do not exceed 4.5%.

In the second time, one does a direct calculation on physical basis, with elements 3D (modelings C, D, E and F). The operators tested are: `DYNA_NON_LINE`, `DEFI_CONTACT` with the methods `CONSTRAINT`, `LAGRANGE` and `CONTINUOUS`.

## 1 Problem of reference

### 1.1 Geometry



Length of the beams:  $L = 1\text{ m}$   
Side of the section of the beams:  $a = 2\text{ cm}$

### 1.2 Material properties

Beam:

Young modulus	$E = 2.10^{11}\text{ Pa}$
Poisson's ratio	$\nu = 0$ for modeling 1D and $0.3$ for modeling 3D
Density	$\rho = 7800\text{ kg/m}^3$

### 1.3 Boundary conditions and loadings

The problem is one-way according to  $x$ .

The beam  $CD$  is embedded in  $D$ , the beam  $AB$  is completely free in translation according to  $x$ .

### 1.4 Initial conditions

With all the nodes of the beam  $AB$  are imposed according to the axis  $x$ :

- an initial speed:  $v_0 = -1\text{ m/s}$

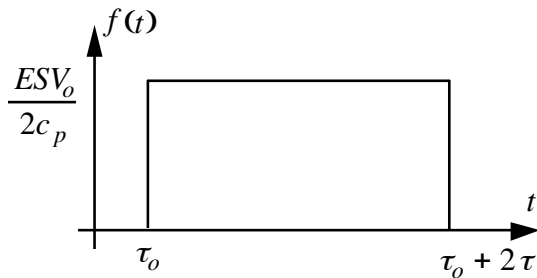
Nodes of the beam  $CD$  a speed and an initial displacement have no one.

Points  $A$  and  $C$  are separated from an initial game  $J_0$  very weak:  $J_0 = 10^{-5}\text{ m}$ .

## 2 Reference solution

### 2.1 Method of calculating

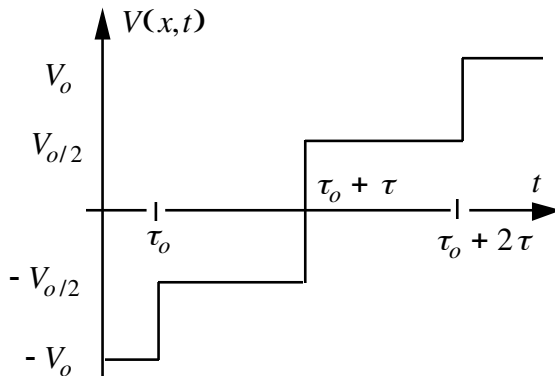
Drawn from [bib1].



$f(t)$  : force de contact en A ;

$V(x, t)$  : vitesse ;

$U(x, t)$  : déplacement ;

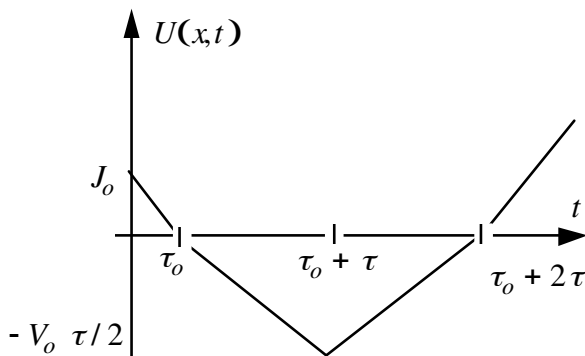


$$\tau_0 = \frac{J_0}{V_0} ;$$

$$\tau = \frac{2L}{c_p} \text{ Durée de choc} = 2 \cdot \tau ;$$

$$c_p = \sqrt{\frac{E(1-\nu)}{\rho(1+\nu)(1-2\nu)}} ;$$

$$S = a^2 \text{ section}$$



..... pour point A

### 2.2 Uncertainty on the solution

No (analytical solution).

### 2.3 Bibliographical reference

- 1) Algorithms of fast dynamics theoretical Description and examples of applications. Report EDF/DER HP-61/93.115

## 3 Modeling A

### 3.1 Characteristics of modeling

Discretization of the two beams by meshes SEG2 (50 each one) and of the finite elements of type POU\_D\_T.

A modal base of 40 clean modes (20 by beams) is used for the modal superposition.  
A contractual reduced modal damping by 0.1% is applied to each clean mode.

The conditions initial speeds are imposed by building a field on the nodes of displacement and by projecting this field on the nodes on the modal basis.  
The vector generalized thus calculated can be introduced into the order DYNA\_TRAN\_MODAL behind the keyword VITE\_INIT\_GENE.

The parameters of modeling of the law of shock used are:

- Stiffness of shock: RIGI\_NOR  $5.10^9 N/m$
- Damping of shock: AMOR\_NOR  $2.10^4 Ns/m$

The first calculation:

- the normal with the plan of the shock is selected according to  $Z : \text{NORM\_OBST} = (0. 1. )$
- an obstacle of the type BI\_PLAN\_Z is selected

The second calculation:

- the normal with the plan of the shock is selected according to  $Y : \text{NORM\_OBST} = (0. 1. 0. )$
- an obstacle of the type BI\_PLAN\_Y is selected

The third calculation:

- the normal with the plan of the shock is selected according to  $Y : \text{NORM\_OBST} = (0. 1. 0. )$
- an obstacle of the type BI\_CERCLE is selected

Values of DIST\_1 and DIST\_2 who are fictitious here and only to model the contact are selected equal to  $\text{DIST}_1 = \text{DIST}_2 = J_o/2$  so that there is contact at the beginning of calculation.

Temporal integration is carried out with the algorithm of Euler and a step of time of  $10^{-6} s$ .

### 3.2 Characteristics of the grid

Many nodes: 102

Many meshes and types: 100 SEG2

## 3.3 Sizes tested and results

The first calculation

Identification	Value of reference	Type of reference	Tolerance
<i>DX</i> at the point <i>A</i> t=2.0e-4 S	- 1.E-4	'ANALYTICAL'	1.0%
<i>DX</i> at the point <i>A</i> t=4.0e-4 S	- 2.E-4	'ANALYTICAL'	3.5%
<i>DX</i> at the point <i>A</i> t=6.0e-4 S	- 1.E-4	'ANALYTICAL'	5.0%
<i>DX</i> at the point <i>A</i> t=8.0e-4 S	-1.E-9	'ANALYTICAL'	1,0E-5
<i>DX</i> at the point <i>A</i> t=1.0e-3 S	2.E-4	'ANALYTICAL'	3.5%

The second calculation

Identification	Value of reference	Type of reference	Tolerance
<i>DX</i> at the point <i>A</i> t=2.0e-4 S	- 1.E-4	'ANALYTICAL'	1.0%
<i>DX</i> at the point <i>A</i> t=4.0e-4 S	- 2.E-4	'ANALYTICAL'	3.5%
<i>DX</i> at the point <i>A</i> t=6.0e-4 S	- 1.E-4	'ANALYTICAL'	5.0%
<i>DX</i> at the point <i>A</i> t=8.0e-4 S	-1.E-9	'ANALYTICAL'	1,0E-5
<i>DX</i> at the point <i>A</i> t=1.0e-3 S	2.E-4	'ANALYTICAL'	3.5%

The third calculation

Identification	Value of reference	Type of reference	Tolerance
<i>DX</i> at the point <i>A</i> t=2.0e-4 S	- 1.E-4	'ANALYTICAL'	1.0%
<i>DX</i> at the point <i>A</i> t=4.0e-4 S	- 2.E-4	'ANALYTICAL'	3.5%
<i>DX</i> at the point <i>A</i> t=6.0e-4 S	- 1.E-4	'ANALYTICAL'	5.0%
<i>DX</i> at the point <i>A</i> t=8.0e-4 S	-1.E-9	'ANALYTICAL'	1,0E-5
<i>DX</i> at the point <i>A</i> t=1.0e-3 S	2.E-4	'ANALYTICAL'	3.5%

## 4 Modeling B

### 4.1 Characteristics of modeling

Discretization of the two beams by meshes SEG2 (50 each one) and of the finite elements of type POU\_D\_T.

A modal base of 40 clean modes (20 by beams) is used for the modal superposition.  
A contractual reduced modal damping by 0.1% is applied to each clean mode.

The conditions initial speeds are imposed by building an initial field speed applied to the beams *poutre1* and *poutre2*.

The parameters of modeling of the law of shock used are:

- the normal with the plan of the shock is selected according to  $Z : \text{NORM\_OBST} : (0.1. 0. )$
- an obstacle of the type BI\_CERC\_INT is selected
- Stiffness of shock: RIGI\_NOR  $5.10^9 \text{ N/m}$
- Damping of shock: AMOR\_NOR  $2.10^4 \text{ Ns/m}$

Temporal integration is carried out with the algorithm of Euler and a step of time of  $10^{-6} \text{ s}$ .

### 4.2 Characteristics of the grid

Many nodes: 102

Many meshes and types: 100 SEG2

### 4.3 Sizes tested and results

Identification	Value of reference	Type of reference	Tolerance
<i>DX</i> at the point <i>A</i> t=2.0e-4 S	- 1.E-4	'ANALYTICAL'	1.0%
<i>DX</i> at the point <i>A</i> t=4.0e-4 S	- 2.E-4	'ANALYTICAL'	3.5%
<i>DX</i> at the point <i>A</i> t=6.0e-4 S	- 1.E-4	'ANALYTICAL'	5.0%
<i>DX</i> at the point <i>A</i> t=8.0e-4 S	-1.E-9	'ANALYTICAL'	1,0E-5
<i>DX</i> at the point <i>A</i> t=1.0e-3 S	1.E-4	'ANALYTICAL'	7.0%
<i>DX</i> at the point <i>A</i> t=1.2e-3 S	2.E-4	'ANALYTICAL'	5.0%
<i>DX</i> at the point <i>A</i> t=1.4e-3 S	1.E-4	'ANALYTICAL'	7.0%
<i>DX</i> at the point <i>A</i> t=1.6e-3 S	-1.E-9	'ANALYTICAL'	1,0E-5

## 5 Modeling C

### 5.1 Characteristics of modeling

The two beams are modelled with meshes HEXA8 (50 per beam) and of the finite elements 3D.  
The behavior is elastic.

The conditions initial speeds are imposed by building a field initial speed applied to the two beams:  $DZ = -1.0$  for POU1 and  $DZ = 0.0$  for POU2.

The shock is modelled by a condition of contact. The method used is CONSTRAINT.

Temporal integration is carried out with the method of modified average acceleration (keyword HHT with  $\alpha = -0,1$ ) and a step of time of  $10^{-6}$  s .

One tests then another algorithm of temporal integration:  $\alpha$  - method (keyword HHT with  $\alpha = -0,3$  and MODI\_EQUI=' OUI ') and a step of unchanged time of  $10^{-6}$  s .

### 5.2 Characteristics of the grid

Many nodes: 408

Many meshes and types: 100 HEXA8

### 5.3 Sizes tested and results

The first calculation

Identification	Value of reference	Type of reference	Tolerance
<i>DZ</i> at the point <i>A</i> t=2.0e-4 S	- 1.050E-4	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=4.0e-4 S	- 1.550E-4	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=6.0e-4 S	- 5.540E-5	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=8.0e-4 S	9.920E-5	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=1.0e-3 S	2.990E-4	'ANALYTICAL'	5.0%

The second calculation

Identification	Value of reference	Type of reference	Tolerance
<i>DZ</i> at the point <i>A</i> t=2.0e-4 S	- 1.050E-4	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=4.0e-4 S	- 1.550E-4	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=6.0e-4 S	- 5.540E-5	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=8.0e-4 S	9.920E-5	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=1.0e-3 S	2.990E-4	'ANALYTICAL'	5.0%

## 6 Modeling D

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### 6.1 Characteristics of modeling

The two beams are modelled with meshes `HEXA8` (50 per beam) and of the finite elements 3D.  
The behavior is elastic.

The conditions initial speeds are imposed by building a field initial speed applied to the two beams: `DZ` = -1.0 for `POU1` and `DZ` = 0.0 for `POU2`.

The shock is modelled by a condition of contact. The method used is `LAGRANGE`.

Temporal integration is carried out with the method `HHT` ( $\alpha = -0.1$ ) and a step of time of  $10^{-6}$  s.

### 6.2 Characteristics of the grid

Many nodes: 408

Many meshes and types: 100 `HEXA8`

### 6.3 Sizes tested and results

Identification	Value of reference	Type of reference	Tolerance
<i>DZ</i> at the point <i>A</i> t=2.0e-4 S	- 1.050E-4	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=4.0e-4 S	- 1.550E-4	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=6.0e-4 S	- 5.540E-5	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=8.0e-4 S	9.920E-5	'ANALYTICAL'	5.0%
<i>DZ</i> at the point <i>A</i> t=1.0e-3 S	2.990E-4	'ANALYTICAL'	5.0%



## 7 Modeling E

### 7.1 Characteristics of modeling

The two beams are modelled with meshes `HEXA8` (50 per beam) and of the finite elements 3D.  
The behavior is elastic.

The conditions initial speeds are imposed by building a field initial speed applied to the two beams: `DZ`  
= -1.0 for `POU1` and `DZ` = 0.0 for `POU2`.

The shock is modelled by a condition of contact. The formulation used is `CONTINUOUS`.

Temporal integration is carried out with the method `HHT` ( $\alpha = -0.1$ ) and a step of time of  $10^{-6}$  s.

### 7.2 Characteristics of the grid

Many nodes: 408

Many meshes and types: 100 `HEXA8`

### 7.3 Sizes tested and results

Identification	Value of reference	Type of reference	Tolerance
<i>DZ</i> at the point <i>A</i> t=2.0e-4 S	- 1.050E-4	'ANALYTICAL'	0.01%
<i>DZ</i> at the point <i>A</i> t=4.0e-4 S	- 1.550E-4	'ANALYTICAL'	0.03%
<i>DZ</i> at the point <i>A</i> t=6.0e-4 S	- 5.540E-5	'ANALYTICAL'	0.2%
<i>DZ</i> at the point <i>A</i> t=8.0e-4 S	9.920E-5	'ANALYTICAL'	6.0%
<i>DZ</i> at the point <i>A</i> t=1.0e-3 S	2.990E-4	'ANALYTICAL'	2.0%

## 8 Modeling F

### 8.1 Characteristics of modeling

The two beams are modelled with meshes `HEXA8` (50 per beam) and of the finite elements 3D.  
The behavior is elastic.

The conditions initial speeds are imposed by building a field initial speed applied to the two beams: `DZ` = -1.0 for `POU1` and `DZ` = +1.0 for `POU2`.

The shock is modelled by a condition of contact. The formulation used is `CONTINUOUS`.

Temporal integration is carried out with the diagram `THETA_METHODE` ( $\theta=0,7$ , formulation in displacement) and a step of time of  $5,0 \times 10^{-6}$  s.

### 8.2 Characteristics of the grid

Many nodes: 408

Many meshes and types: 100 `HEXA8`

### 8.3 Sizes tested and results

Identification	Value of reference	Type of reference	Tolerance
<i>DZ</i> at the point <i>A</i> t=0.5e-5 S	-0.5E-5	'ANALYTICAL'	0.1%
<i>DZ</i> at the point <i>A</i> t=1.5e-5 S	-1.0E-5	'ANALYTICAL'	0.1%
<i>DZ</i> at the point <i>A</i> t=2.5e-4 S	-1.0E-5	'ANALYTICAL'	0.1%
<i>DZ</i> at the point <i>A</i> t=3.4e-4 S	-1.0E-5	'ANALYTICAL'	0.1%
<i>DZ</i> at the point <i>A</i> t=5.0e-4 S	1.0	'ANALYTICAL'	0.1%

## 9 Summary of the results

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For modelings A and B (with `DYNA_TRAN_MODAL`):

The precision of calculation is relatively average what is due to the choice of the coefficients of penalization used to model the contact. The increase in the stiffness of contact improves considerably the field of displacement but generates important oscillations of the field speed around the analytical solution.

For modelings C, D and E (with `DYNA_NON_LINE`):

The precision of calculation is very good (4% of maximum change). In this case, the three methods used give results of comparable quality.

Moreover, for modeling C, one also tested two types of diagrams of integration in implicit times: modified average acceleration (keyword `HHT` with the option `MODI_EQUI='NON'` : option by default) and “complete” HHT (keyword `HHT` with the option `MODI_EQUI='OUI'`).

With “complete” diagram HHT, the maximum variation observed with the reference solution drops slightly: 2.15% compared with 3.73% with the modified average acceleration. The other values tested are impacted very little, with the choice of values of the parameter  $\alpha$  diagrams employed in this CAS-test ( $\alpha=0.1$  for the modified average acceleration and  $\alpha=0.3$  for HHT).