Titre : FDLV115 – Réponse harmonique d'un anneau visco-éla[...] Responsable: ABBAS Mickaël

Date: 11/12/2017 Page: 1/5 Clé: V8.01.115 Révision

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# FDLV115 - Harmonic answer of a viscoelastic ring in coupling fluid-structure for the reduction of model

### **Summary:**

The objective of this test is of to check in an elementary way the operator of reduction of model DEFI BASE REDUITE on a case of YEWS with a viscoelastic material.

Titre : FDLV115 – Réponse harmonique d'un anneau visco-éla[...]

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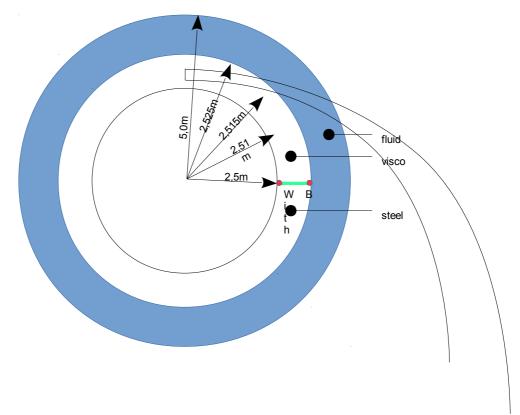
Révision 069b66ef1e7c

Date: 11/12/2017 Page: 2/5

### 1 Problem of reference

## 1.1 Geometry

One is considered metal ring plan plunged in a fluid environment with an external border for this last (figure 1.1-1).



In the metal

## 1.2 Properti

Figure 1.1-1: Geometry (not on the scale)

For the part out of steel, one considers an isotropic elastic material:

- Modulus of elasticity:  $E_a = 177 \, GPa$
- Poisson's ratio:  $v_a = 0.3$
- Density:  $\rho_a = 7450 \, kg \cdot m^{-3}$

For the fluid, it is water with the following characteristics:

- Speed of sound:  $c=1500 \, m.s^{-1}$
- Density:  $\rho_{s} = 1000 \, kg \, . \, m^{-3}$

For the viscoelastic part, one considers material of a model type of the fractional derivative (model of Zener) whose modulus of rigidity depends on the pulsation:

$$G^{*}(\omega) = \frac{G_{0} + G_{\infty}(i\omega\tau)^{\alpha}}{1 + (i\omega\tau)^{\alpha}}$$
 (1)

The characteristics are the following ones:

- Poisson's ratio:  $v_{\nu} = 0.49$  (quasi-incompressible material)
- Density:  $\rho_{v} = 1460 \, kg \, . \, m^{-3}$
- Coefficients:  $G_0$ =2,11 MPa ,  $G_{\infty}$ =0,59 GPa ,  $\tau$ =0,44 $\times$ 10<sup>-6</sup> and  $\alpha$ =0,53





Titre : FDLV115 – Réponse harmonique d'un anneau visco-éla[...] Responsable : ABBAS Mickaël Date : 11/12/2017 Page : 3/5 Clé : V8.01.115 Révision

069b66ef1e7c

Note: to improve calculation of the modes, one carries out the have-dimensioning of all these quantities compared to water.

### 1.3 Boundary conditions and loadings

For the structure part, one imposes a condition of embedding on the segment AB and a horizontal nodal force  $F_X = A .0 \times 10^{-5} N$  on the point A.

A condition of interaction fluid-border is carried out by the use of elements <code>2D\_FLUI\_STRU</code> with the interface between Lhas structure and fluid. In addition, a condition of impedance is defined on the border external of the fluid, by a loading of the type <code>IMPE</code> and the calculation of the matrices of impedance which are dependent there.

Titre: FDLV115 – Réponse harmonique d'un anneau visco-éla[...] Responsable : ABBAS Mickaël

Clé: V8.01.115 Révision

Date: 11/12/2017 Page: 4/5

069b66ef1e7c

#### 2 Reference solution

#### 2.1 Method of calculating

The empirical modes are calculated. One tests the modes by specific of not-regression on various points and different values ddl.

#### 2.2 Sizes and results of reference

One seeks to solve the following system:

$$\left(-i\omega^{3}\boldsymbol{I}(\mu)-\omega^{2}(\mu)\boldsymbol{M}(\mu)+\boldsymbol{K}(\mu)\right)\boldsymbol{u}(\mu)=\boldsymbol{F}(\mu)$$
(2)

represent the parameters of variation, I the matrix of impedance, M the matrix K the matrix of rigidity and F the second member (applying the nodal force). The variable parameters are the following:  $\,\omega\,$  ,  $\,E_{a}\,$  ,  $\,\rho_{a}\,$  and  $\,\rho_{v}\,$  . One writes the system in the following form:

$$f_1(\omega, E_a). \mathbf{K}_v + f_2(\omega, \rho_a). \mathbf{K}_e + \mathbf{K}_f + f_3(\omega, \rho_v). \mathbf{M}_e - \omega^2 \mathbf{M}_f - j \omega^3. \mathbf{I} = \mathbf{F}$$
(3)

Different the contribution was separated:  $K_{\nu}$  for rigidity coming from the viscoelastic part (calculated by <code>RIGI\_MECA\_HYST</code>),  $m{K}_e$  for rigidity coming from the elastic part,  $m{K}_f$  for rigidity coming from the fluid  $M_e$  for the mass coming from the elastic part,  $M_f$  for the mass coming from the fluid part and I the matrix of impedance.

And functions:

$$f_{1}(\omega, E_{a}) = 2 \times E_{a}(1.0 + v_{v}) \frac{\frac{G_{0} + G_{\infty}(jc \omega \tau)^{\alpha}}{1 + (jc \omega \tau)^{\alpha}}}{\rho_{e} c^{2}} + 2j E_{a}(1.0 + v_{v}) \frac{\frac{G_{0} + G_{\infty}(jc \omega \tau)^{\alpha}}{1 + (jc \omega \tau)^{\alpha}}}{\rho_{e} c^{2}} \times \frac{\Im\left(\frac{G_{0} + G_{\infty}(jc \omega \tau)^{\alpha}}{1 + (jc \omega \tau)^{\alpha}}\right)}{\Re\left(\frac{G_{0} + G_{\infty}(jc \omega \tau)^{\alpha}}{1 + (jc \omega \tau)^{\alpha}}\right)}$$

$$(4)$$

$$f_2(\omega, \rho_a) = \frac{\rho_a E_e}{\rho_a c^2} \tag{5}$$

$$f_3(\omega, \rho_v) = \frac{-\omega^2 \rho_v \rho_a}{\rho_e} \tag{6}$$

For calculation, one varies the four parameters out of ten values (one does not recopy here the value of these parameters which were drawn randomly, to see the command file)

#### 2.3 Uncertainties on the solution

The error on the solution depends on the degree of reduction (many modes empiricalS).

Titre : FDLV115 – Réponse harmonique d'un anneau visco-éla[...]

Date: 11/12/2017 Page: 5/5 Responsable: ABBAS Mickaël Clé: V8.01.115

Révision 069b66ef1e7c

#### **Modeling A** 3

#### 3.1 **Characteristics of modeling**

A modeling is used 2D FLUIDE, D PLAN and 2D FLUI STRU in linear dynamics YEWS coupled (formulation  $(u, p, \varphi)$ ).

This modeling tests the creation of the empirical modes. Three modes are calculated.

#### 3.2 Characteristics of the grid

The grid contains 10654 elements of the type TRIA3.

#### 3.3 Sizes tested and results

One tests some values (complex) of the base:

Identification	Type of reference
N 5 - D X - Mode 1	NON_REGRESSION
N 5 - DY - Mode 1	NON_REGRESSION
N 9 - NEAR - Mode 1	NON_REGRESSION
N 5 - D X - Mode 2	NON_REGRESSION
N 5 - DY - Mode 2	NON_REGRESSION
N9 - NEAR - Mode 2	NON_REGRESSION
N 5 - D X - Mode 3	NON_REGRESSION
N 5 - DY - Mode 3	NON_REGRESSION
N 9 - NEAR - Mode 3	NON_REGRESSION

#### Remarks 3.4

One cannot say anything in the absolute on the precision of these values because ON tests values of not-regression. It is a purely data-processing test.