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# SSLS133 - Flexbeam with variable thickness

### Abstract:

This test represents a quasi-static computation of a flexbeam with variable thickness. It is embedded at an end, and is subjected to a vertical force at the other end. This test makes it possible to test shell elements voluminal SHB8 and SHB20 to manage the variations of thickness. Four modelizations are tested:

Finite elements SHB8 for a linear variation of the thickness of the plate (modelization A ). Finite elements SHB20 for a linear variation of the thickness of the plate (modelization B ). Finite elements SHB8 for a quadratic variation of the thickness of the plate (modelization C ). Finite elements SHB20 for a quadratic variation of the thickness of the plate (modelization D).

Displacements obtained are compared with the elastic analytical solution of a beam in bending. This test makes it possible to show the capacities and the limiting of elements SHB8 and SHB20 to manage the variations of thickness.

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## 1 Problem of reference

## 1.1 Geometry

### 1.1.1 Plates with thickness varying linearly

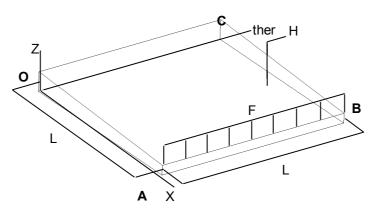


Figure 1.1.1-1:

Length L=100 m, width l=100 m.

The thickness h varies linearly:

$$h(x) = ax + b$$

We pose h(x=0)=hI=10m and h(x=L)=h2=5m what gives us:

$$a = (h_2 - h_1)/L$$
 and  $b = h_1$ 

## 1.1.2 Plates with thickness varying quadratically

the thickness h varies in a quadratic way:

$$h(x) = ax^2 + bx + c$$

We pose  $h(x=0)=hI=10\,m$  ,  $h(x=L)=h2=5\,m$  and  $h(x=L/2)=h_{12}=6,25\,m$  what gives us

$$a = (2(h_1 + h_2) - 4h_{12})/L^2$$
,  $b = (4h_{12} - h_2 - 3h_1)/L$  and  $c = h_1$ 

# 1.2 Material properties

Modulus Young:  $E = 2.10^{11} Pa$ 

Poisson's ratio: v = 0.0

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# 1.3 Boundary conditions and loadings

Boundary conditions:

Embedded on the side OC : u=v=w=0 ,  $\theta_x=\theta_v=\theta_z=0$ 

Loading:

At the end AB , a load uniformly distributed of resultant:

Force parallel with the axis Z;  $F_z = 1 N$ 

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#### Reference solution 2

#### 2.1 Method of calculating used for the reference solution

the results of reference are got by the theory of the elastic beams.

In the case of a linear variation of the thickness, vertical displacement at the end ABis given by [1]:

$$w(x) = -\frac{FL^{2}}{2 EI_{y,c}^{2}} \frac{\left(2 Lcx + c^{2} x^{2} - c^{3} x^{2} + 2L(L + cx) \ln\left(\frac{L}{L + cx}\right)\right)}{(L + cx)}$$

With

$$c = \left(\frac{I_{y_2}}{I_{y_1}}\right)^{\frac{1}{3}} - 1$$
 and  $I_{y_i} = \frac{bh_i^3}{12}$ 

In the case of a quadratic variation of the thickness, it is possible to find a formula exact of displacement. However its general statement is sufficiently complex not to be able to be written here. We formulated the approximate function of vertical displacement according to x our case:

$$w(x) = 3.10^{-8} \frac{2x - 200}{x^2 - 200x + 20000} + 6.10^{-10} \arctan(0.01x - 1) - 3.10^{-12}x + 7.71238.10^{-10} m$$

#### 2.2 Quantities and results of reference

Displacement of the points A and B following Z.

#### 2.3 Bibliographical references

[1] [V3.01.400] SSLL400 – non-prismatic Beam, subjected to forces specific or distributed.

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## 3 Modelization A

### 3.1 Characteristic of the modelization

Element SHB8 and thickness varying linearly

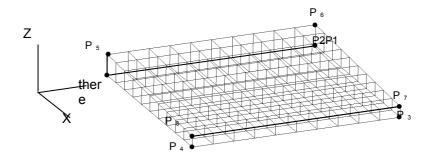


Figure 3.1-1: Mesh of the modelization A

Cutting: a regular mesh is considered in this modelization.

Regular mesh:

100 meshes SHB8: 10 according to the width, 10 according to the length, 1 according to the thickness

Boundary conditions:

All nodes inside the side  $P_1P_2P_6P_5$ : blocked displacement following X All the nodes on the edge  $P_1P_5$ : blocked displacement following Y All the nodes on the edge  $P_2P_1$ : blocked displacement following Z

Loading:

Force linearly distributed on the edge  $P_8P_7$  : F = 1

Name of the nodes:

Not P1	N022	Not P5	N020
Not P2	N002	Not P6	N001
Not P3	N102	Not P7	N100
Not P4	N172	Not P8	N171

## 3.2 Characteristics of the mesh

Many nodes: 242 Number of meshes and types: 100 SHB8

# 3.3 Quantities tested and regular

### Mesh results:

Not	Quantity in unit	Reference	Aster	% difference
P7	displacement W ( m	)3.2710 10-10	3.2711 10-10	+0.004
P8	displacement W ( m	)3.2710 10-10	3.2711 10-10	+0.004

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# **Modelization B**

#### 4.1 Characteristic of the modelization

Element SHB20 and thickness varying linearly

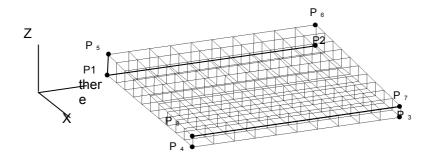


Figure 4.1-1: Mesh of the modelization B

Cutting: a regular mesh is considered in this modelization.

Regular mesh:

100 meshes SHB20: 10 according to the width, 10 according to the length, 1 according to the

thickness

Boundary conditions:

All nodes inside the side  $P_1P_2P_6P_5$ : blocked displacement following X All the nodes on the edge  $P_1P_5$ : blocked displacement following Y All the nodes on the edge  $P_2P_1$ : blocked displacement following Z

Loading:

Force linearly distributed on the edge  $P_8P_7$ : F=1

Name of the nodes:

Not P1	N347	Not P5	N340
Not P2	N579	Not P6	N572
Not P3	N006	Not P7	N002
Not P4	N074	Not P8	N067

#### Characteristics of the mesh 4.2

Many nodes: 803 Number of meshes and types: 100 SHB20

#### 4.3 Quantities tested and regular

### Mesh results:

	Not	Quantity in unit	Reference	Aster	% difference
_	P7	displacement W ( m	)3.2710 10-10	3.2866 10-10	+0.476
	P8	displacement W ( m	)3.2710 10-10	3.2866 10-10	+0.476

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#### **Modelization C** 5

#### 5.1 Characteristic of the modelization

Element SHB8 and thickness varying quadratically

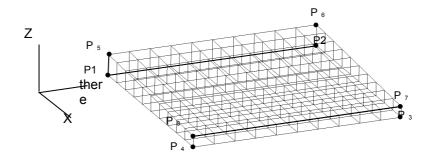


Figure 5.1-1: Mesh of the modelization C

the characteristics are the same ones as for the modelization A

Name of the nodes:

Not P1	N005	Not P5	N003
Not P2	N006	Not P6	N004
Not P3	N008	Not P7	N002
Not P4	N007	Not P8	N001

#### 5.2 Characteristics of the mesh

The mesh is the same one as the modelization Aexcept for the thickness which varies here in a quadratic way.

### 5.3 Quantities tested and regular

### Mesh results:

Not	Quantity in unit	Reference	Aster	% difference	
P7	displacement W ( m	)4.7124 10-10	5.1212 10-10	+8.67	
P8	displacement W ( m	)4.7124 10-10	5.1212 10-10	+8.67	

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#### Modelization D 6

#### 6.1 Characteristic of the modelization

Element SHB20 and thickness varying quadratically

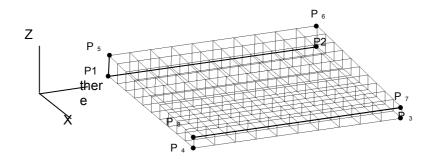


Figure 6.1-1: Mesh of the modelization D

the characteristics are the same ones as for the modelization B

Name of the nodes:

Not P1	N005	Not P5	N003
Not P2	N006	Not P6	N004
Not P3	N008	Not P7	N002
Not P4	N007	Not P8	N001

#### 6.2 Characteristics of the mesh

The mesh is the same one as the modelization Bexcept for the thickness which varies here in a quadratic way.

#### 6.3 Quantities tested and regular

### Mesh results:

No	Quantity in unit	Reference	Aster	% difference
P7	displacement W ( m	)4.7124 10-10	4.6754 10-10	-0.784
P8	displacement W ( m	)4.7124 10-10	4.6754 10-10	-0.784

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# Summary of the results

In the case in a variation linaire of the thickness of the plate, good solutions are obtained some is the finite element used (SHB8 or SHB20).

When the geometrical variation is of a quadratic nature, elements SHB20 provide more precise results (error <1% ) that elements SHB8 (error <9% ).