

## SSLS117 - Eccentering of Summarized asymmetric

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### plates:

This test validates the eccentering of asymmetric simple plates compared to the plane of the mesh or plane of diagram (key word `EXCENTREMENT` of the command `AFFE_CARA_ELEM`).

The reference is given by a first resolution where one models double-layered made up of two layers of various thickness and two materials.

It is used to validate the second computation where one models two layers offset compared to the plane of the mesh.

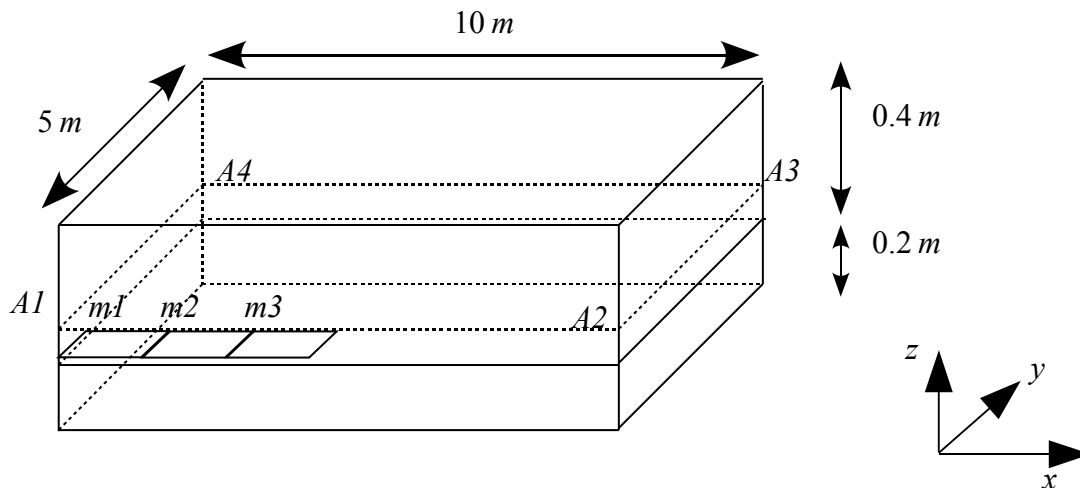
It differs from test `SSLS111` only by the fact that the 2 layers are different thickness.

Four modelizations implement elements `DKT`, `DKQ`, `DST`, `DSQ`.

One highlights on the double-layered case the difference between the modelizations, due to the effect of shears transverse.

## 1 Problem of reference

### 1.1 Geometry



### 1.2 Properties of the materials

#### 1.2.1 Modelizations A and B

the material has an isotropic elastic behavior:

Young's modulus:  $E = 20000 \text{ MPa}$

Poisson's ratio:  $\nu = 0$ .

Density:  $\rho = 1000 \text{ kg/m}^3$

#### 1.2.2 Modelizations C and D

the material is double-layered.

The material constituting the first layer is elastic orthotropic and is characterized by the following data:

$$E_L = 20000 \text{ MPa}$$

$$E_T = 20000 \text{ MPa}$$

$$\nu_{LT} = 0.3$$

$$G_{LT} = 2000 \text{ MPa}$$

The material constituting the second layer is also elastic orthotropic and is characterized by the following data:

$$E_L = 15000 \text{ MPa}$$

$$E_T = 15000 \text{ MPa}$$

$$\nu_{LT} = 0.3$$

$$G_{LT} = 1500 \text{ MPa}$$

### 1.3 Boundary conditions and loadings

#### 1.3.1 Modelizations A and B

the edge  $A1A4$  is clamped.

One applies a distributed force  $F_z = -1000 \text{ N/m}$  to the edge  $A2A3$ .

## 1.3.2 Modelizations C and D

the node  $A1$  is clamped  $DX = DY = DZ = 0.$   
 $DRX = DRY = DRZ = 0.$

the node  $A2$  is blocked according to the following degrees of freedom:  
 $DX = DY = 0.$

One applies the nodal forces  $FZ = -1000 N$  to the node  $A3$ , and one applies the distributed loading (key word `FORCE_COQUE`) to meshes  $m1$ ,  $m2$  and  $m3$  :

$$FX = 200 N \quad FY = -500. N/m^2 \quad FZ = -500. N/m^2 \quad MX = 100. N/m \quad MY = 40. N/m$$

The selected loading utilizes requests out of membrane and bending.

## 2 Reference solution

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### 2.1 Method of calculating used for the reference solution

The computation with the double-layered material (command `DEFI_COMPOSITE`) is used as reference. The non regression one compared to the results got by this first computation is checked. The two plates of the second modelization are offset compared to the average plane of the double-layered one.

### 2.2 Results of reference

#### 2.2.1 Modelizations A and B

They are made up by the values of the field of displacement  $DZ$  at the point  $A3$  and the forces generalized with the node  $AI$ . In addition, the 4 smaller frequencies of structure are calculated.

#### 2.2.2 Modelization C and D

They are made up by the values of the field of displacement  $DX, DY, DZ, DRX, DRY$  at the point  $A3$  and the point  $A4$ .

One compares also the forces generalized with the node  $AI$ .

In addition, the 4 smaller frequencies of structure are calculated.

### 2.3 Uncertainty on the solution

For the modelizations  $A$  and  $B$ , the reference solution is analytical. There is thus no uncertainty.

For the modelizations  $C$  and  $D$ , uncertainties are null since it is about the same computation carried out by two different ways.

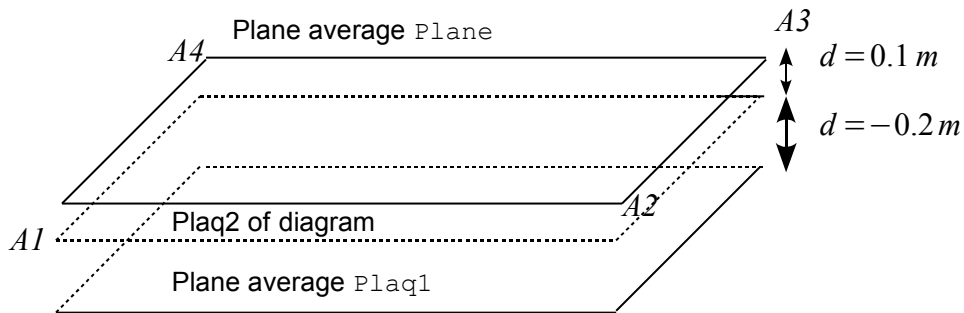
### 2.4 Bibliographical references

- 1) [R3.07.03]: Shell elements `DKT, DST, DKQ, DSQ` and `Q4G`.
- 2) [R3.07.06]: Processing of the eccentricity for shell elements `DKT, DST, DKQ, DSQ` and `Q4G`.

## 3 Modelization A

### 3.1 Characteristic of the modelization

The model consists of two plates corresponding to the average plane of each of the two layers of the model of reference. To represent these two plates, one leaves the mesh of the plane of diagram which one offsets distances  $0.1\text{ m}$  and  $-0.2\text{ m}$ .



the elements used are shell elements DKT.

### 3.2 Characteristics of the mesh

Coordinated of the nodes:

Node	X	Y	Z
$A1 = N65$	0.	0.	0.
$A2 = N66$	10.	0.	0.
$A3 = N1$	10.	5.	0.
$A4 = N51$	0.5	.	0.

- 94 nodes; GROUP\_MA:  $L14 = A1A4$ ,  $L12 = A2A3$
- 100 meshes DKT (TRIA3).

### 3.3 Quantities tested and Standard

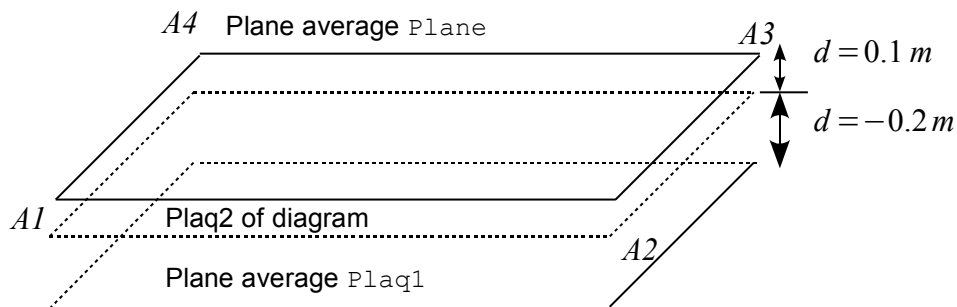
Identification	results of reference	Reference	Tolerance (%)
<b>Displacement</b>			
DZ ( A3 )	"ANALYTIQUE"	$-9.259259 \cdot 10^{-5}$	0.5
<b>Eigenfrequencies</b>			
Frequency 1st mode	"NON_REGRESSION"	13.67	0.1
Frequency 2nd mode	"NON_REGRESSION"	64.86	0.1
Frequency 3rd mode	"NON_REGRESSION"	84.47	0.1
Frequency 4th mode	"NON_REGRESSION"	101.16	0.1
<b>generalized Forces</b>			
$MXX$ N66	"ANALYTIQUE"	5000.	0.1
$QX$ N66	"ANALYTIQUE"	- 500.	0.1

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## 4 Modelization B

### 4.1 Characteristic of the modelization

The model consists of two plates corresponding to the average plane of each of the two layers of the model of reference. To represent these two plates, one leaves the mesh of the plane of diagram which one offsets distances  $0.1\text{ m}$  and  $-0.2\text{ m}$ .



the elements used are shell elements DKQ.

### 4.2 Characteristics of the mesh

Coordinated of the nodes:

Node	X	Y	Z
A1	0.	0.	0.
A2	10.	0.	0.
A3	10.	5.	0.
A4	0.5	.	0.

- 67 nodes
- 50 meshes DKQ (QUAD4)

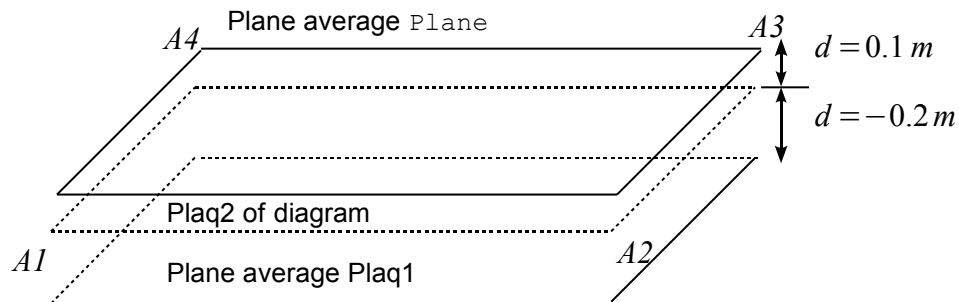
### 4.3 Quantities tested and Standard

Identification	results of Reference	Reference	Tolerance (%)
<b>Displacement</b>			
DZ ( A3 )	"ANALYTIQUE"	$-9.259259 \cdot 10^{-5}$	0.5
<b>Eigenfrequencies</b>			
Frequency 1st mode	"NON_REGRESSION"	13.7	0.1
Frequency 2nd mode	"NON_REGRESSION"	66.65	0.1
Frequency 3rd mode	"NON_REGRESSION"	85.49	0.1
Frequency 4th mode	"NON_REGRESSION"	100.1	0.1
<b>generalized Forces</b>			
MXX N60	"ANALYTIQUE"	5000.	0.1
QX N60	"ANALYTIQUE"	- 500.	0.1

## 5 Modelization C

### 5.1 Characteristic of the modelization

The model consists of two plates corresponding to the average plane of each of the two layers of the model of reference. To represent these two plates, one leaves the mesh of the plane of diagram which one offsets distances  $0.1 m$  and  $-0.2 m$ .



the elements used are shell elements DST.

### 5.2 Characteristics of the mesh

Coordinated of the nodes:

Node	X	Y	Z
A1 = N1	0.	0.	0.
A2 = N11	10.	0.	0.
A3 = N65	10.	5.	0.
A4 = N66	0.5	.	0.

- 66 Nodes
- 100 meshes DST (TRIA3)

## 5.3 Quantities tested and Standard

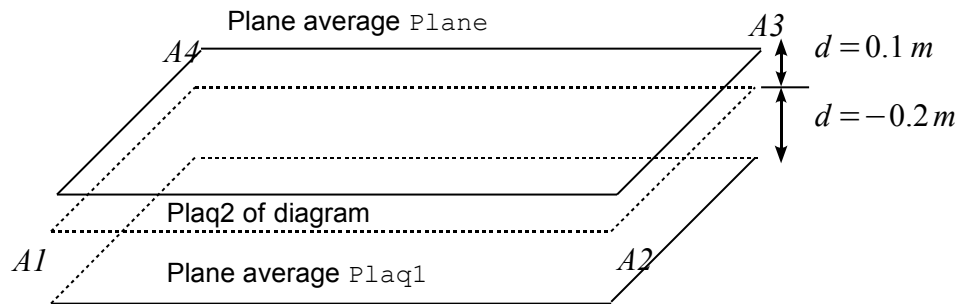
Identification	results of Reference	Reference	Tolerance (%)
<b>Displacement</b>			
DX ( A4 )	"NON_REGRESSION"	- 1.939 10 <sup>-6</sup>	0.1
DY ( A4 )	"NON_REGRESSION"	- 1.149 10 <sup>-6</sup>	0.1
DZ ( A4 )	"NON_REGRESSION"	- 2.2091 10 <sup>-4</sup>	0.6
DRX ( A4 )	"NON_REGRESSION"	- 6.09302 10 <sup>-5</sup>	0.6
DRY ( A4 )	"NON_REGRESSION"	1.297279 10 <sup>-4</sup>	0.4
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DX ( A3 )	"NON_REGRESSION"	- 2.4385 10 <sup>-6</sup>	0.1
DY ( A3 )	"NON_REGRESSION"	- 2.3382 10 <sup>-7</sup>	0.5
DZ ( A3 )	"NON_REGRESSION"	- 1.5864 10 <sup>-3</sup>	0.4
DRX ( A3 )	"NON_REGRESSION"	- 1.2639 10 <sup>-4</sup>	0.3
DRY ( A3 )	"NON_REGRESSION"	1.4127 10 <sup>-4</sup>	0.4
<hr/>			
<b>Eigenfrequencies</b>			
Frequency 1st mode	"NON_REGRESSION"	1.512356	0.1
Frequency 2nd mode	"NON_REGRESSION"	6.373398	0.1
Frequency 3rd mode	"NON_REGRESSION"	1.25011 101.0.1	
Frequency 4th mode	"NON_REGRESSION"	2.546726 101.0.1	
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<b>generalized Forces</b>			
NXX AI	"NON_REGRESSION"	9.85902 E+03	0.3
NYX AI	"NON_REGRESSION"	6.36055 E+03	0.5
NXY AI	"NON_REGRESSION"	2.07601 E+03	1.5
MXX AI	"NON_REGRESSION"	2.11639 E+04	0.5
MYX AI	"NON_REGRESSION"	1.49410 E+04	0.5
MXZ AI	"NON_REGRESSION"	5.82623 E+03	0.65
QX AI	"NON_REGRESSION"	- 2.56538 E+03	0.6
QY AI	"NON_REGRESSION"	1.79286 E+03	1.5



## 6 Modelization D

### 6.1 Characteristic of the modelization

The model consist of two plates corresponding to the average plane of each of the two layers of the model of reference. To represent these two plates, one leaves the mesh of the plane of diagram which one offsets distances  $0.1 m$  and  $-0.2 m$ .



the elements used are shell elements DSQ.

### 6.2 Characteristics of the mesh

Coordinated of the nodes:

Node	X	Y	Z
A1 = N1	0.	0.	0.
A2 = N51	10.	0.	0.
A3 = N65	10.	5.	0.
A4 = N66	0.5	.	0.

- 67 nodes
- 50 meshes DSQ (QUAD4)

## 6.3 Quantities tested and Standard

Identification	results of reference	Reference	Tolerance (%)
<b>Displacement</b>			
DX ( A4 )	"NON_REGRESSION"	- 2.34539 10 <sup>-6</sup>	0.1
DY ( A4 )	"NON_REGRESSION"	- 1.9694 10 <sup>-6</sup>	0.1
DZ ( A4 )	"NON_REGRESSION"	- 2.2428 10 <sup>-4</sup>	0.1
DRX ( A4 )	"NON_REGRESSION"	- 6.2983 10 <sup>-5</sup>	0.1
DRY ( A4 )	"NON_REGRESSION"	1.5823 10 <sup>-4</sup>	0.1
DX ( A3 )	"NON_REGRESSION"	- 3.0023 10 <sup>-6</sup>	0.1
DY ( A3 )	"NON_REGRESSION"	-4.6612 10 <sup>-7</sup>	0.1
DZ ( A3 )	"NON_REGRESSION"	- 1.8842 10 <sup>-3</sup>	0.1
DRX ( A3 )	"NON_REGRESSION"	- 1.2768 10 <sup>-4</sup>	0.1
DRY ( A3 )	"NON_REGRESSION"	1.7064 10 <sup>-4</sup>	0.1
<b>Eigenfrequencies</b>			
Frequency 1st mode	"NON_REGRESSION"	1.4219	0.1
Frequency 2nd mode	"NON_REGRESSION"	5.2995	0.1
Frequency 3rd mode	"NON_REGRESSION"	1.215 101.0.1	
Frequency 4th mode	"NON_REGRESSION"	2.4385 101.0.1	
<b>generalized Forces</b>			
NXX AI	"NON_REGRESSION"	8.68372 E+03	0.1
NYY AI	"NON_REGRESSION"	- 4.10693 E+03	0.1
NXY AI	"NON_REGRESSION"	3.90190 E+02	0.1
MXX AI	"NON_REGRESSION"	3.47663 E+04	0.1
MYX AI	"NON_REGRESSION"	1.52451 E+04	0.1
MYX AI	"NON_REGRESSION"	6.34489 E+03	0.1
QX AI	"NON_REGRESSION"	- 1.70439 E+04	0.1
QY AI	"NON_REGRESSION"	- 9.82819 E+03	0.1

## 7 Summary of the results

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With regard to displacements for modelizations `DKT` and `DKQ`, the results obtained with 2 offset shells differ from to the more 2% compared to the reference solution. For the other modelizations, one obtains to the maximum of the errors of 4% for the `DST` and 6% for the `DSQ`. For these two last modelizations, the error is more important because the computation of the transverse shears is not equivalent between double-layered and the two offset plates.

Indeed, the transverse shears are supposed to be constant in the thickness of each element `DST` or `DSQ` ; these transverse shears are average shears. One thus obtains a mean value for each plate offset, overall different from the average transverse shears on the double-layered plate.

This is marked even more for the forces, where the differences `A` remain lower than 5% for the modelizations `DKT` ( and `B`) but reach 11% for the modelization `C` and 100% for modelization `D`.

One can thus note that for the `DSQ`, 2 computations (double-layered and full-course offset) are far from being equivalent. One preserves nevertheless this modelization precisely to display this difference.