

## SSLP305 - Thin disc out of bearing under concentrated loading

---

### Summarized:

The purpose of the test is validating the computation of potential energy in linear elasticity.

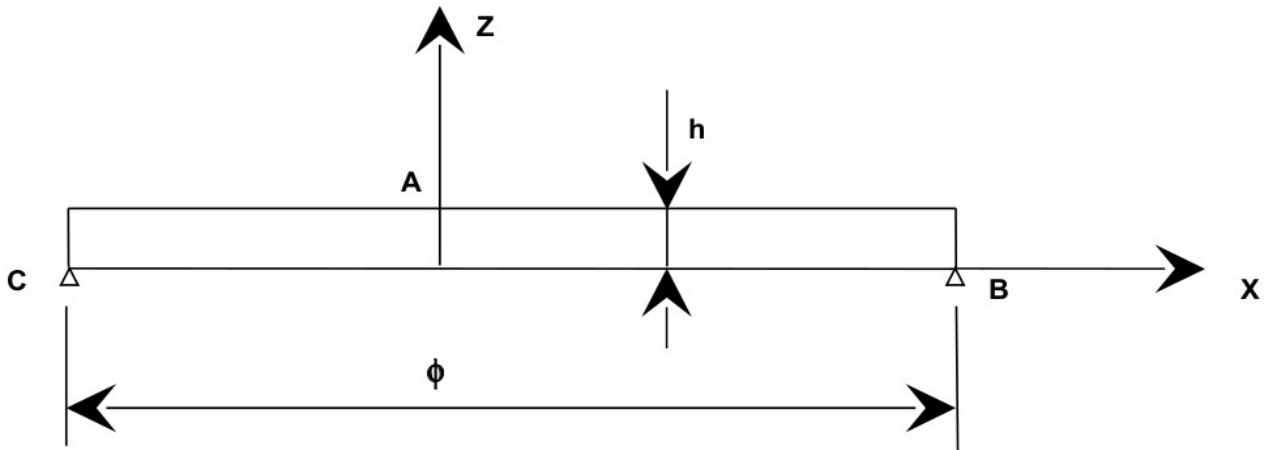
Only one axisymmetric modelization is presented.

The reference solution is analytical.

## 1 Problem of reference

---

### 1.1 Geometry



Diameter:  $\phi = 0.5 \text{ m}$

Thickness:  $h = 0.005 \text{ m}$

### 1.2 Material properties

Modulus Young:  $E = 2.1 \times 10^{11} \text{ Pa}$

Poisson's ratio:  $\nu = 0.3$

### 1.3 Boundary conditions and loadings

- Bearing on edge (  $w = 0$  )
- Charges concentrated with point:  $A \quad P = -350 \text{ N}$

### 1.4 Initial conditions

Without object for the static analysis.

## 2 Reference solution

---

### 2.1 Method of calculating used for the reference solution

- the value of axial displacement to the center of the disc (not A) is given by:

$$W_a = -\frac{P \phi^2}{64 \pi D} \times \frac{3 + \nu}{1 + \nu}$$

where  $D = \frac{E h^3}{12(1 - \nu^2)}$

- the value of potential energy (with the equilibrium) is given by:

$$E_p = -\frac{1}{2} P W_a$$

- The absolute value of potential energy by radian is:

$$e_p = \frac{1}{2} \frac{P W_a}{2 \pi}$$

### 2.2 Results of reference

- Displacement to point: A  $W_a = -0.4596 \times 10^{-3} m$
- Potential energy by radian:  $e_p = 0.012799 Nm / rd$

### 2.3 Uncertainty on the analytical

solution Solution.

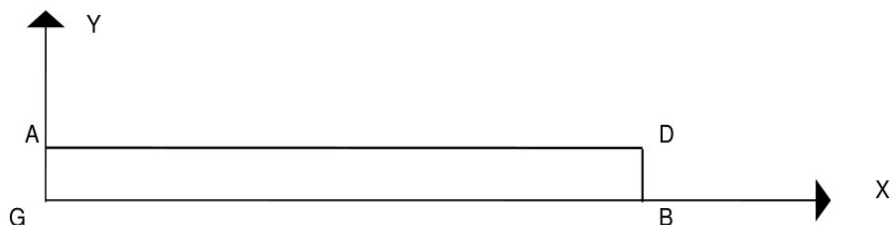
### 2.4 Bibliographical references

- 1) R.J. ROARK and W.C. YOUNG Formulated for stress and strain, 5th edition, New York, Mc Graw-Hill, 1975

## 3 Modelization A

### 3.1 Characteristic of the modelization

It is an axisymmetric modelization.



Limiting conditions:

in  $B$                     DDL\_IMPO: (GROUP\_NO: B        DY: 0.)  
on  $AG$                     DDL\_IMPO: (GROUP\_NO: 1AG      DX: 0.)

Loading:

in  $A$                     FORCE\_NODALE: (GROUP\_NO: A    FY: -55.704)

Name of the nodes:

$A = N1$                  $B = N755$              $D = N858$              $G = N201$

Cutting:                100 elements according to the radius  
                             2 elements according to the thickness

### 3.2 Characteristics of the mesh

Many nodes: 905

Number of meshes and types: 100 QUADS 8,200 SORTED 6,208 SEG3

### 3.3 Values tested

Standard	Localization of value	Reference	Aster	% difference
Not $A$	$W_A(m)$	$-0.4596 \cdot 10^{-3}$	$-0.4617 \cdot 10^{-3}$	0.46
	$e_p(Nm/rd)$	$-1.2799 \cdot 10^{-2}$	$-1.2859 \cdot 10^{-2}$	0.47

### 3.4 Remarks

- the value of the load required is brought back to a sector of 1 radian. Consequently, the value of the potential energy given on the results file corresponds to the strain of this sector (with the sign near).
- Option ENERPOT calculates in fact a strain energy:

Warning : The translation process used on this website is a "Machine Translation". It may be imprecise and inaccurate in whole or in part and is provided as a convenience.

$E_d = \frac{1}{2} U^T K U$  who is identical to potential energy with the sign near:

$$E_p = \frac{1}{2} U^T K U - U^T F = -\frac{1}{2} U^T F = -\frac{1}{2} U^T K U \text{ (because } KU = F \text{)}$$

## 4 Summary of the results

---

These good performances on displacement and strain energy (similar variation of 0,5% with the analytical reference solution) show that the computation of this energy is correct. To approach still best the value of reference, the mesh would have to be discretized more.