

SSLP01 – Shears and flexbeam in its Summarized

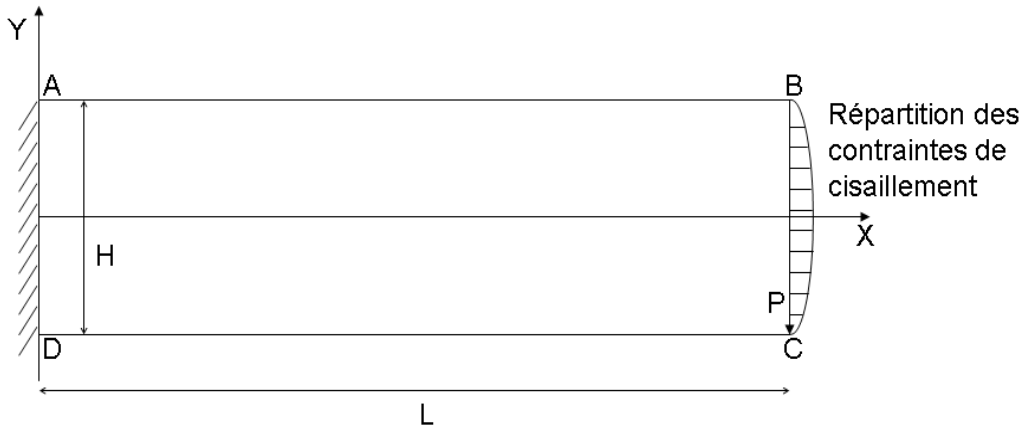
plane:

In this benchmark one models the behavior of a shears and flexbeam in his plane.

Only one modelization is carried out: C_PLAN

1 Problem of reference

1.1 Geometry



Coordinated of the points (*m*):

$$A : (0., 6 \cdot 10^{-3})$$

$$B : (48 \cdot 10^{-3}, 6 \cdot 10^{-3})$$

$$C : (48 \cdot 10^{-3}, -6 \cdot 10^{-3})$$

$$D : (0., -6 \cdot 10^{-3})$$

Geometry of the plate (*m*):

Thickness: $h = 0,001$
Width: $L = 0.048$
Height: $H = 0.012$

Mesh group: *BORD_CH* surface of right (*BC*)

Mesh group: *ENCAST* surface of left (*AD*)

Mesh group: *SURF* surface intern

1.2 Properties of the material

- $E = 3 \cdot 10^{10} Pa$
- $\nu = 0.25$

1.3 Boundary conditions and loadings

- imposed Displacement:
 - *ENCAST* : $DX = DY = 0.$
- Loading:
 - Parabolic distribution on the height, constant on the thickness.

$Y (m)$	-0,006	-0,003	0	0,003	0,006
Shearing stress 2D (<i>Pa.m</i>)	0	3.75E6	5.00E6	3.75E6	0

integration of this stress on the height H leads to a stress resulting from $80.10^3 Pa.m$ which one notes P in what follows.

2 Reference solution

2.1 Method of calculating

result of reference was obtained by analytical computation with the method of the functions of Airy.

- Plane stresses:

$$\begin{aligned}\sigma_{xx} &= (12.P.y.(x-L))/2.H^3 \\ \sigma_{yy} &= 0 \\ \sigma_{xy} &= 6.P.((H^2/4)-y^2)/2.H^3\end{aligned}$$

- Displacements:

$$\begin{aligned}u &= \frac{12P}{EhH^3} \left[y \left(\frac{x^2}{2} - Lx \right) - \left(1 + \frac{\nu}{2} \right) \frac{y^3}{3} \right] + Ay + B \\ v &= \frac{-12P\nu}{EhH^3} \frac{y^2}{2} (x-L) + \frac{12P}{EhH^3} \left[-\frac{x^3}{3} + \frac{Lx^2}{2} + (1+\nu) \frac{H^2 x}{4} \right] - Ax + C\end{aligned}$$

- The constants A , B , C depend on the boundary conditions on displacements:

$$\begin{aligned}u(0,0) &= v(0,0) = \frac{\partial v}{\partial x}(0,0) = 0 \\ u\left(0, -\frac{H}{2}\right) &= v\left(0, -\frac{H}{2}\right) = u\left(0, \frac{H}{2}\right) = v\left(0, \frac{H}{2}\right) = 0\end{aligned}$$

2.2 Results of reference

Displacement according to y point: $x=L; y=0$ $v=0.3413 \cdot 10^{-3}$ m

Stress according to x with point: $x=0; y=-H/2$ $\sigma_{xx}=80 \cdot 10^6$ Pa

2.3 Uncertainties

analytical Solution

3 Modelization A

3.1 Characteristic of the modelization

Modelization C_PLAN :

Many nodes	177			
Number of meshes	80	Are:		
			SEG3	32
			QUAD8	48

3.2 Results

Points	Quantity	Reference	Tolerance (relative)
$x=L; y=0$	DY	$3,41 \cdot 10^{-3} m$	0.023
$x=0; y=-H/2$	$SIXX$	$80 \cdot 10^6 Pa$	0.015

4 Summary of the results

the results obtained in displacement and stress with modelization C_PLAN are satisfactory.