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## SSLL118 – Clamped beam subjected to the displacements defined in a Summarized

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### local coordinate system:

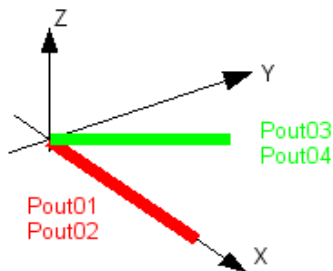
This case test makes it possible of the command to validate `THE OPTION DDL_POUTRE AFFE_CHAR_MECA`, which makes it possible to impose the displacements defined in a local coordinate system related to the beam.

The conditions on the degrees of freedom  $(dx, dy, dz, drx, dry, drz)$  expressed in the local coordinate system are translated into conditions on the degrees of freedom  $(DX, DY, DZ, DRX, DRY, DRZ)$  expressed in the total reference.

This case test in particular makes it possible to validate the fact, that to impose a null displacement in a local coordinate system is correctly translated in the total reference.

## 1 Problems of reference

### 1.1 Geometry



There are 4 beams length  $L$  modelled with `POU_D_E`.

*Pout01* *Pout02* : they are along the total axis  $X$ .

*Pout03* *Pout04* : their axis is in the direction  $(1.0, 1.0, 0.0)$ .

### 1.2 Properties of the material

Characteristic of the material

$$E = 2.0E11\text{Pa} \quad \nu = 0.30$$

### 1.3 geometrical Characteristics

All the beams have the same rectangular section:

$$HY = 0.2 \quad HZ = 0.1$$

The length of the beams is of 2m .

### 1.4 Boundary conditions and loadings

All the beams are embedded in the beginning: nodes  $N1, N2, N3, N4$ .

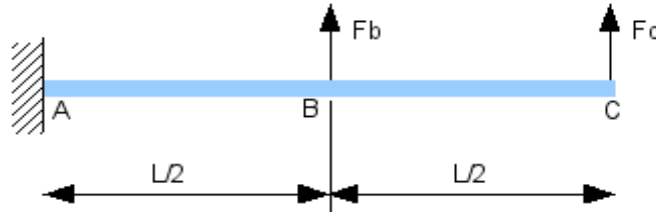
A displacement is imposed at the other end: nodes  $NA, NB, NC, ND$ .

A force can be imposed on the medium node: nodes  $IA, IB, IC, ID$ .

## 2 Reference solution

### 2.1 Method of calculating

the reference solution uses the results below:



$\delta_c$  and  $\delta_b$  represent displacement at the points  $C$  and  $B$ .

$$EI \cdot \delta_c = \frac{F_c \cdot L}{3} + \frac{5 \cdot F_b \cdot L^3}{48} \quad [1]$$

$$EI \cdot \delta_b = \frac{5 \cdot F_c \cdot L^3}{47} + \frac{F_b \cdot L^3}{24} \quad [2]$$

$$\text{the reaction of bearing } F_a = -F_c - F_b \quad [3]$$

the boundary conditions which is used during the validation are: a displacement in  $C$  :  $\delta_c$  and a possible force in  $B$  :  $F_b$ . The resolution of the equations [1], [2], [3] give the following statements:

$$\delta_b = \frac{7 \cdot F_b \cdot L^3}{768 \cdot EI} + \frac{5 \cdot \delta_c}{16}$$

$$F_c = \frac{3 \cdot \delta_c \cdot EI}{L^3} - \frac{5 \cdot F_b}{16}$$

$$F_a = -\frac{3 \cdot \delta_c \cdot EI}{L^3} - \frac{11 \cdot F_b}{16}$$

### 2.2 Quantities and results of reference

the computations carried out on the 4 beams differ either by boundary conditions imposed on their nodes  $B$  and  $C$ , or by a definition different from the local coordinate system. The boundary conditions imposed in displacement or force make it possible to determine the theoretical solutions of the various cases considered.

Characteristics of beam:

$$3 \cdot EI_y = 10E+07 \text{ N.m}^2 \quad 3 \cdot EI_z = 4.0E+07 \text{ N.m}^2$$

Name of beam: *Pout01*

Data:

$$\text{VECT\_Y} : (-0.0 \ 1.0 \ 0.0)$$

$$\delta_y \text{ in } C \text{ (in the local coordinate system)} : 2.0E-03 \text{ m}$$

$$\delta_z \text{ in } C \text{ (in the local coordinate system)} : 1.0E-03 \text{ m}$$

Theoretical results:

$$F_Y \text{ in } A \text{ (in the total reference)} : \frac{-3 \cdot \delta_y \cdot EI_z}{L^3}$$

$$FZ \text{ in } A \text{ (in the total reference)} : \frac{-3 \cdot \delta z \cdot EIy}{L^3}$$

Name of beam: *Pout02*

Data:

$$\text{VECT\_Y} : (-0.0 \ 0.0 \ 1.0)$$

$$\delta y \text{ in } C \text{ (in the local coordinate system)} : 2.0E-03 \text{ m}$$

$$\delta z \text{ in } C \text{ (in the local coordinate system)} : 1.0E-03 \text{ m}$$

Theoretical results:

$$FY \text{ in } A \text{ (in the total reference)} : \frac{3 \cdot \delta z \cdot EIz}{L^3}$$

$$FZ \text{ in } A \text{ (in the total reference)} : \frac{-3 \cdot \delta y \cdot EIy}{L^3}$$

Name of beam: *Pout03*

Data:

$$\text{VECT\_Y} : (-1.0 \ 1.0 \ 0.0)$$

$$\delta y \text{ in } C \text{ (in the local coordinate system)} : 2.0E-03 \text{ m}$$

$$\delta z \text{ in } C \text{ (in the local coordinate system)} : 1.0E-03 \text{ m}$$

Theoretical results:

$$FX \text{ in } A \text{ (in the total reference)} : \frac{-3\sqrt{2} \cdot \delta y \cdot EIz}{2 \cdot L^3}$$

$$FY \text{ in } A \text{ (in the total reference)} : \frac{3\sqrt{2} \cdot \delta y \cdot EIz}{2 \cdot L^3}$$

$$FZ \text{ in } A \text{ (in the total reference)} : \frac{-3 \cdot \delta z \cdot EIy}{L^3}$$

Name of beam: *Pout04*

Data:

$$\text{VECT\_Y} : (-1.0 \ 1.0 \ 0.0)$$

$$\delta y \text{ in } C \text{ (in the local coordinate system)} : 2.0E-03 \text{ m}$$

$$\delta z \text{ in } C \text{ (in the local coordinate system)} : 0$$

$$FZ \text{ in } B \text{ (in the total reference)} : 1000.0$$

Theoretical results:

$$FX \text{ in } A \text{ (in the total reference)} : \frac{-3\sqrt{2} \cdot \delta y \cdot EIz}{2 \cdot L^3}$$

$$FY \text{ in } A \text{ (in the total reference)} : \frac{-3\sqrt{2} \cdot \delta y \cdot EIz}{2 \cdot L^3}$$

$$FZ \text{ in } A \text{ (in the total reference)} : \frac{-11 \cdot Fb}{16}$$

$$DZ \text{ in } B \text{ (in the total reference)} : \frac{7 \cdot Fb \cdot L^3}{768 \cdot EI}$$

## 3 Modelization A

### 3.1 Characteristic of the mesh and the modelization

Many nodes : 12  
Many SEG2 : 8  
Number of mesh group : The 4

beams are modelled with POU\_D\_E.

### 3.2 Quantities tested and results

the quantities tested are the reactions to the fixed support and the displacement of the node *ID* .

Standard	beam of the field	Component	Node	Values of reference	Relative Error
<i>Pout01</i>	FORC_NODA	DY	<i>N1</i>	-10000.0	0.0%
		DZ	<i>N1</i>	-1250.0	0.0%
<i>Pout02</i>	FORC_NODA	DY	<i>N2</i>	5000.0	0.0%
		DZ	<i>N2</i>	-2500.0	0.0%
<i>Pout03</i>	FORC_NODA	DX	<i>N3</i>	7071.0678	1.34E- 05%
		DY	<i>N3</i>	-7071.0678	1.34E- 05%
		DZ	<i>N3</i>	-1250.0	1.32E- 05%
<i>Pout04</i>	FORC_NODA	DX	<i>N4</i>	7071.0678	1.34E- 05%
		DY	<i>N4</i>	-7071.0678	1.34E- 05%
		DZ	<i>N4</i>	-687.5	5.79E- 06%
	DEPL	DZ	<i>ID</i>	2.1875E-05	-1.93E- 05%

## 4 Modelization B

### 4.1 Characteristic of the mesh and the modelization

Many nodes : 16  
Number of SEG2 : 12  
Number of mesh group : The 6

beams are modelled with POU\_D\_E.

### 4.2 Quantities tested and results

the quantities tested are the reactions to the fixed support and the displacement of the node *ID* .

Standard	beam of the field	Standard	Component	Node of Reference	Values of reference	Tolerance
<i>Pout01</i>	FORC_NODA	DY	<i>N1</i>	"ANALYTIQUE"	-10000.0	0,1%
		DZ	<i>N1</i>	"ANALYTIQUE"	-1250.0	0,1%
<i>Pout02</i>	FORC_NODA	DY	<i>N2</i>	"ANALYTIQUE"	5000.0	0,1%
		DZ	<i>N2</i>	"ANALYTIQUE"	-2500.0	0,1%
<i>Pout03</i>	FORC_NODA	DX	<i>N3</i>	"ANALYTIQUE"	7071.0678	0,1%
		DY	<i>N3</i>	"ANALYTIQUE"	-7071.0678	0,1%
		DZ	<i>N3</i>	"ANALYTIQUE"	1250.0	0,1%
<i>Pout04</i>	FORC_NODA	DX	<i>N4</i>	"ANALYTIQUE"	7071.0678	0,1%
		DY	<i>N4</i>	"ANALYTIQUE"	-7071.0678	0,1%
		DZ	<i>N4</i>	"ANALYTIQUE"	-687.5	0,1%
	DEPL	DZ	<i>ID</i>	"ANALYTIQUE"	2.1875E-05	0,1%

### 4.3 Remarks

This modelization makes it possible to validate OF THE COMMAND DDL\_POUTRE AFPE\_CHAR\_MECA with the key word MAILLE/GROUP\_MA, the beams *Pout01* , *Pout02* , *Pout03* , *Pout04* were lengthened of 1m.

The place of application of the boundary conditions and the loadings did not change.

## 5 Summary of the results

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This case test makes it possible to validate the use of `DDL_POUTRE` of the command `AFFE_CHAR_MECA`. One validates in particular the fact that to impose a null displacement in a local coordinate system is correctly translated in the total reference.